## IMPLICIT LARGE EDDY SIMULATIONS OF 2D FLOW AND HEAT TRANSFER IN THERMOACOUSTIC RESONATORS

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### Introduction:

In this study, we have devised a numerical 2D model (plane) symmetry) of a linear thermoacoustic machine composed of two heat exchangers, stack and a resonator. It solves the coupled compressible Navier-Stokes and heat transfer equations using COMSOL Multiphysics software, using an ideal gas model for air, and P2-P1 finite elements.



Implicit Large Eddy Simulation (iLES) has been used to simulate the transitional flow and thermoacoustic dynamics in this device.

These techniques are numerically efficient, since they require only a moderate refinement of the spatial mesh and time-stepping sequence, without any additional terms or equations [4].

#### **Mathematical Model:**

The evolution of the physical parameters is governed by the Navier-Stokes equations applied to the gas circulating in the resonator and in the stack, using the ideal gas model for air.

> $\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho u) = 0$ аı

thermoacoustic wave generator

Mesh used in the stack

X: 0.6626

Y: -1478

**Results**: The calculation is done using a triangular mesh with a total cell number of 25523 (~ 180 000 dofs)







pressure in the stack

$$\rho \stackrel{\sim}{=} + \rho(u \cdot \nabla) u = \nabla [-p I + \mu(\nabla u + (\nabla u^{2}))]$$

$$\frac{\partial}{\partial t} = -\frac{2}{3} \mu(\nabla \cdot u) I + F$$

$$\rho C_{p} \frac{\partial T}{\partial t} + \rho C_{p} u \cdot \nabla T = \nabla \cdot (k \nabla T) + Q + Q_{vh} + W_{p}$$

Valeurs

With :

 $W_p = \alpha_p T \left( \frac{\partial p_A}{\partial t} \right)$  $\alpha_p = -\frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right),$ 

Initial and boundary conditions for air (ideal gas model) :

$$(e_1^{0})_{0}^{0}$$
  $(e_1^{0})_{0}^{0}$   $(e_1$ 

Fourier transformation

#### **Conclusion:**

This confirms that the results of the COMSOL numerical model of this thermoacoustic device match a linear thermoacoustic machine [1-2], in which the same frequency is obtained.

The simulation was carried out by solving the mass conservation equation, momentum and energy equations, for compressible flows.

Paramètres

Initial tomorroture (1/)

Initial temperature (K)	300
Ambient pressure (Pa)	101325
Inlet pressure to the engine (Pa)	10
Heat transfer coefficient in the stack (W/m^2	2/K) 50
Initial velocity (m/s)	0
Time step (s) : maximum 10 <sup>-5</sup>	initial: 10 <sup>-7</sup>

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